

# Fundamentals and recent advances in vehicle platooning control

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**CHALMERS**

# Lecture content

- ① Main Concepts in ACC
- ② Control design in vehicle following mode
  - ① requirements
  - ② constant spacing
  - ③ constant time gap
- ③ Spacing error propagation in constant spacing policies
  - ① Predecessor following control scheme
  - ② Predecessor and leader following control scheme
- ④ Disturbance propagation
- ⑤ String stability in heterogenous platoons
  - ① Predecessor following control scheme
  - ② Predecessor and leader following control scheme

**Note:** The following notes have been extracted from

- “Vehicle Dynamics and Control” by R. Rajamani
- “Disturbance Propagation in Vehicle Strings”, P. Seiler, A. Pant, K. Hendrick. *IEEE Transactions on Automatic Control*, vol. 49, no. 10, October 2004
- “Controller Design for String Stable Heterogeneous Vehicle Strings”, E. Shaw, K. Hendrick. *46<sup>th</sup> IEEE Conference on Decision and Control*, 2007

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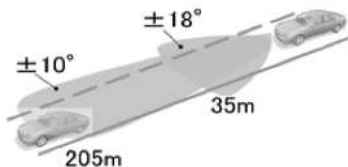
# Main concepts and intro

## ACC objectives

- ➊ Maintaining a constant vehicle longitudinal speed in absence of preceding vehicles
- ➋ Maintaining a “safe” distance from the preceding (slower) vehicle, if any

## Actuators

- ➊ Engine
- ➋ Brakes



## Sensors

- ➊ Speed sensor (odometer)
- ➋ Radar
  - ▶ Range through reflections
  - ▶ Range rate through doppler effect

**Note.** The ACC is an “autonomous” system. I.e., no wireless

# Main concepts and intro

- First introduced in Japan in early nineties
- Originally thought as a “comfort and convenience” system
- According statistics (over 90% highways accident cause by human errors<sup>1</sup>) may impact safety as well
- Basis of many automated driving systems available on the market

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<sup>1</sup>US Department of Transportation, 1992

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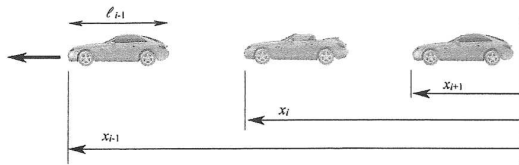
# Vehicle following. Control requirements

## Individual vehicle stability

## String stability

The *string stability* property implies that, during velocity transients, the non-zero spacing errors do not amplify toward the tail of a string of ACC vehicles<sup>a</sup>

<sup>a</sup>Swaroop, 1995, Swaroop and Hedrick, 1996



Individual vehicle stability is trivial. *We will focus on string stability*



# Vehicle following. Vehicle model

## Assumptions

- Two level hierarchical control
- Upper level calculates a desired acceleration to meet the control requirements
- Lower level calculates the engine and brake low level control inputs

Hence, model the  $i$ -th vehicle as either a double integrator

$$\ddot{x}_i = u_i,$$

or as

$$\ddot{x}_i = \frac{e^{-s\tau}}{a + sT} u_i,$$

where  $u_i = \ddot{x}_{i_{des}}$ . Typically,

$$-5m/s^2 \leq \ddot{x} \leq 2m/s^2$$

# Vehicle following. String stability

## Definition

Define

$$H(s) = \frac{\delta_i(s)}{\delta_{i-1}(s)}.$$

The chain of ACC vehicles is string stable if<sup>a</sup>

- ❶  $\|H(s)\|_\infty \leq 1$
- ❷  $h(t) > 0, \forall t \geq 0$

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<sup>a</sup>Swaroop, 1995

Intuitively,

- ❶ Condition 1 guarantees that  $\|\delta_i\|_2 \leq \|\delta_{i-1}\|_2$
- ❷ Condition 2 implies that the *steady state* spacing errors have the same sign

More rigorous explanation follows

## ...short detour to norms for signals and systems

### Definitions (signals)

Consider a signal  $u(t) : t \in [-\infty, \infty] \rightarrow u \in \mathbb{R}$ . Define the following norms

❶ **1-Norm**  $\|u\|_1 = \int_{-\infty}^{\infty} |u(t)| dt$

❷ **2-Norm**  $\|u\|_2 = \left( \int_{-\infty}^{\infty} u(t)^2 dt \right)^{1/2}$

❸  **$\infty$ -Norm**  $\|u\|_{\infty} = \sup_t |u(t)|$

### Definitions (systems)

Consider a linear, time-invariant, causal system  $y = g * u$ , where  $g$  is the impulse response and  $G = \mathcal{L}(g)$

❶ **2-Norm**  $\|G\|_2 = \left( \frac{1}{2\pi} \int_{-\infty}^{\infty} |G(j\omega)|^2 d\omega \right)^{1/2}$

❷  **$\infty$ -Norm**  $\|G\|_{\infty} = \sup_{\omega} |G(j\omega)|$

# Useful results on gains<sup>2</sup>

## 2-norm/2-norm gain

Consider the system  $y = g * u$ , with  $G = \mathcal{L}(g)$ .

$$\|G\|_{\infty} = \sup \frac{\|y\|_2}{\|u\|_2}$$

## Proof.

By the Parseval's theorem

$$\begin{aligned}\|y\|_2^2 &= \|Y\|_2^2 = \frac{1}{2\pi} \int_{-\infty}^{\infty} |G(j\omega)|^2 |U(j\omega)|^2 d\omega \\ &\leq \|G\|_{\infty}^2 \frac{1}{2\pi} \int_{-\infty}^{\infty} |U(j\omega)|^2 d\omega \\ &= \|G\|_{\infty}^2 \|U\|_2^2 = \|G\|_{\infty}^2 \|u\|_2^2\end{aligned}$$

Show now that  $\|G\|_{\infty}$  is the least upper bound on the 2-norm/2-norm gain.

— Choose  $u$  such that  $\|u\|_2 = 1$  and show that  $\|Y\|_2^2 = \|G\|_{\infty}^2$  □

# Useful results on gains

## 2-norm/ $\infty$ -norm gain

Consider the system  $y = g * u$ , with  $G = \mathcal{L}(g)$ .

$$\|G\|_2 = \sup \frac{\|y\|_\infty}{\|u\|_2}$$

## Proof.

Apply the Cauchy-Schwarz inequality

$$\begin{aligned} |y(t)| &= \left| \int_{-\infty}^{\infty} g(t - \tau) u(\tau) d\tau \right| \\ &\leq \left( \int_{-\infty}^{\infty} g(t - \tau)^2 d\tau \right)^{1/2} \left( \int_{-\infty}^{\infty} u(\tau)^2 d\tau \right)^{1/2} \\ &= \|g\|_2 \|u\|_2 = \|G\|_2 \|u\|_2 \end{aligned}$$

Hence  $\|y\|_\infty \leq \|G\|_2 \|u\|_2$

Proof follows the same steps as before



## Useful results on gains

### $\infty$ -norm gain/2-norm

Consider the system  $y = g * u$ , with  $G = \mathcal{L}(g)$ .

$$\frac{\|y\|_2}{\|u\|_\infty} = \infty$$

### Proof.

Choose a sinusoidal input signal with frequency  $\omega$ , such that  $\omega$  is not a zero of  $G$ . Hence  $\|u\|_\infty = 1$  and  $\|y\|_2^2$  is unbounded  $\square$

# Useful results on gains

## $\infty$ -norm/ $\infty$ -norm gain

Consider the system  $y = g * u$ , with  $G = \mathcal{L}(g)$ .

$$\|g\|_1 = \sup \frac{\|y\|_\infty}{\|u\|_\infty}$$

Proof.

$$\begin{aligned} |y(t)| &= \left| \int_{-\infty}^{\infty} g(t-\tau)u(\tau)d\tau \right| \leq \int_{-\infty}^{\infty} |g(t-\tau)u(\tau)| d\tau \\ &\leq \int_{-\infty}^{\infty} |g(t-\tau)| d\tau \|u\|_\infty = \|g\|_1 \|u\|_\infty \end{aligned}$$

Hence  $\|y\|_\infty \leq \|g\|_1 \|u\|_\infty$

Proof follows the same steps as before



## Useful results on gains<sup>3</sup>

If  $g(t) > 0 \forall t \geq 0$  then  $\|g\|_1 = \|G\|_\infty$

Proof.

Be  $\gamma_p = \sup \frac{\|y\|_p}{\|u\|_p}$  for a induced  $p$ -norm. Since  $\frac{\|y\|_p}{\|u\|_p} \leq \|g\|_1$ ,

$$|G(0)| \leq \|G(j\omega)\|_\infty \leq \gamma_p \leq \|g\|_1.$$

If  $g(t) > 0$  then

$$|G(0)| = \left| \int_0^\infty g(\tau) d\tau \right| \leq \int_0^\infty |g(\tau)| d\tau = \|g\|_1$$



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<sup>3</sup>Swaroop, 1995



## In summary

Table: System gains

	$\ u\ _2$	$\ u\ _\infty$
$\ y\ _2$	$\ G\ _\infty$	$\infty$
$\ y\ _\infty$	$\ G\ _2$	$\ g\ _1$

Moreover, if  $g(t) > 0 \forall t \geq 0$  then  $\|g\|_1 = \|G\|_\infty$

# Vehicle following. String stability

## Definition

Define

$$H(s) = \frac{\delta_i(s)}{\delta_{i-1}(s)}.$$

The chain of ACC vehicles is string stable if<sup>a</sup>

- ❶  $\|H(s)\|_\infty \leq 1$
- ❷  $h(t) > 0, \forall t \geq 0$

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<sup>a</sup>Swaroop, 1995

The main objective is to obtain

$$\|\delta_i\|_\infty \leq \|\delta_{i-1}\|_\infty,$$

i.e.,  $\|h\|_1 \leq 1$ . This is equivalent to  $\|H\|_\infty \leq 1$ , with the additional condition  $h(t) > 0, \forall t \geq 0$ .

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# Constant spacing control design

Define the *inter-vehicle spacing* as

$$\epsilon_i = x_i - x_{i-1} + \ell_{i-1},$$

where  $\ell_{i-1}$  is the length of the  $(i-1)$ -th vehicle.

Define the *spacing error* as

$$\delta_i = x_i - x_{i-1} + L_{des},$$

where  $L_{des}$  is the desired distance and includes  $\ell_{i-1}$ .

Consider a double integrator model for the vehicle and a *linear PD controller*

$$\ddot{x}_i = -k_p \delta_i - k_v \dot{\delta}_i$$

# Constant spacing control design

Differentiate twice the spacing error

$$\ddot{\delta}_i = \ddot{x}_i - \ddot{x}_{i-1} = -k_p\delta_i - k_v\dot{\delta}_i + k_p\delta_{i-1} + k_v\dot{\delta}_{i-1}$$

Rearranging leads to the closed-loop error dynamics

$$\ddot{\delta}_i + k_v\dot{\delta}_i + k_p\delta_i = k_p\delta_{i-1} + k_v\dot{\delta}_{i-1},$$

corresponding to the transfer function

$$H(s) = \frac{\delta_i(s)}{\delta_{i-1}(s)} = \frac{k_p + k_v s}{s^2 + k_v s + k_p}$$

**Problem.** Find  $k_p$ ,  $k_v$  such that

$$\|H\|_\infty \leq 1$$

# Constant spacing control design

**Solution.** For individual vehicle stability, it must be  $k_v, k_p > 0$ .

Rewrite  $H(s)$  as

$$H(s) = \underbrace{\frac{k_p}{s^2 + k_v s + k_p}}_{H_1(s)} \underbrace{\left( \frac{k_v}{k_p} s + 1 \right)}_{H_2(s)}$$

In order to have  $\|H_1\|_\infty < 1$ , the damping must be larger than 0.707, i.e.,

$$\frac{k_v}{2\sqrt{k_p}} \geq 0.707 \Rightarrow k_v \geq 1.4141\sqrt{k_p}$$

$H_2$  has to be below one up to the resonant frequency  $\sqrt{k_p}$ . Hence,

$$\frac{k_p}{k_v} \geq \sqrt{k_p} \Rightarrow \sqrt{k_p} > k_v$$

## Constant spacing control design

**Solution.** In conclusion, the following conditions have to be satisfied

$$k_v \geq 1.4141\sqrt{k_p}, \quad \sqrt{k_p} > k_v, \quad k_p, k_v > 0$$

*String stability can't be achieved with a PD controller based on **constant spacing policy***

**Question.** Can string stability be achieved with any other **linear controller**?

**Answer.** No, unless. . . .

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# Constant time gap control design

In Constant Time Gap (CTG) control policy, the desired inter-vehicle distance varies with the speed

Define the *spacing error* as

$$\delta_i = x_i - x_{i-1} + L_{des},$$

where  $L_{des} = \ell_{i-1} + h\dot{x}_i$  and  $h$  is the time gap

Consider a double integrator model for the vehicle and the control law

$$u_i = -\frac{1}{h} (\dot{\epsilon}_i + \lambda\delta_i)^4$$

The error dynamics become

$$\dot{\delta}_i = -\lambda\delta_i$$

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<sup>4</sup>Chien, 1993

# Constant time gap control design

Analyze the string stability property of the CTG policy

Combine the first order vehicle model and the control law  $u_i = -\frac{1}{h}(\dot{\epsilon}_i + \lambda\delta_i)$ . Obtain

$$\tau\ddot{x}_i + \ddot{x}_i = -\frac{1}{h}(\dot{\epsilon}_i + \lambda\delta_i)$$

Differentiate twice the spacing error  $\delta_i = \epsilon_i + h\dot{x}_i$  and replace  $\ddot{x}_i$  to obtain

$$\ddot{\epsilon}_i = \ddot{\delta}_i + \frac{1}{\tau}(\dot{\delta}_i + \lambda\delta_i)$$

Solve for  $\epsilon_i$  and replace in  $\delta_i - \delta_{i-1} = \epsilon_i - \epsilon_{i-1} + h\dot{\epsilon}_i$  to obtain

$$H(s) = \frac{\delta_i}{\delta_{i-1}} = \frac{s + \lambda}{h\tau s^3 + hs^2 + (1 + \lambda h) + \lambda}$$

**Problem.** Find condition on  $\tau$  and  $h$  such that  $\|H\|_\infty \leq 1$

# Constant time gap control design

## Theorem

## Proof

Consider the transfer function

$$H(s) = \frac{\delta_i}{\delta_{i-1}} = \frac{s + \lambda}{h\tau s^3 + hs^2 + (1 + \lambda h) + \lambda}$$

Substitute  $s = j\omega$

$$H(s)|_{s=j\omega} = \frac{j\omega + \lambda}{(\lambda - h\omega^2) + j\omega(1 + \lambda h - \tau h\omega^2)}$$

Calculate

$$|H(s)|^2 = \frac{\omega^2 + \lambda^2}{(\lambda - h\omega^2)^2 + \omega^2(1 + \lambda h - \tau h\omega^2)^2}$$

# Constant time gap control design

## Proof (Cont.)

Imposing  $|H(j\omega)| \leq 1$  leads to

$$\omega^2 + \lambda^2 \leq (\lambda - h\omega^2)^2 + \omega^2 (1 + \lambda h - \tau h\omega^2)^2$$

Squaring the terms in parentheses and rearranging

$$\tau^2 h^2 \omega^4 + (h^2 - 2\tau h - 2\tau \lambda h^2) \omega^2 + \lambda^2 h^2 \geq 0$$

Study positiveness of  $a\omega^4 + b\omega^2 + c$ . Rewrite

$$\begin{aligned} a\omega^4 + b\omega^2 + c &= a \left( \omega^4 + 2\frac{b}{2a}\omega^2 + \frac{c}{a} \right) \\ &= a \left[ \left( \omega^2 + \frac{b}{2a} \right)^2 + \frac{4ac - b^2}{4a^2} \right] \end{aligned}$$

# Constant time gap control design

## Proof (Cont.)

Hence  $a\omega^4 + b\omega^2 + c > 0$  if

- ❶  $a, b, c > 0$
- ❷  $b < 0, a > 0, c > 0$  and  $4ac - b^2 > 0$ , i.e.,  $b^2 - 4ac < 0$

Distinguish the following two cases

- ❶  $b > 0$  corresponds to  $h^2 - 2\tau h - 2\lambda\tau h^2 > 0$ . Hence

$$h > \frac{2\tau}{1 - 2\lambda\tau}.$$

For small  $\lambda$ , this is possible if  $h > 2\tau$ .

- ❷  $b < 0, a > 0, c > 0$  and  $b^2 - 4ac < 0$  corresponds to

$$(h^2 - 2\tau - 2\lambda\tau h^2)^2 - 4\tau^2 h^4 \lambda^2$$

# Constant time gap control design

## Proof (Cont.)

2 Simplify to obtain

$$\lambda < \frac{4\tau h - h^2 - 4\tau^2}{8\tau^2 h - 4\tau h^2},$$
$$\lambda < \frac{-(2\tau - h)^2}{4\tau h (2\tau - h)}.$$

Since  $\lambda > 0$ , it must be  $h > 2\tau$ .

By relaxing the inequality in  $a\omega^4 + b\omega^2 + c > 0$ ,  $h \geq 2\tau$  follows.

By 1) and 2) also follows that if  $h \geq 2\tau$  a  $\lambda$  can be found such that  $|H(j\omega)| < 1$ . □

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# Motivation and intro

Recall that for a string of ACC vehicles we have found out that

*String stability can't be achieved with a PD controller based on **constant spacing policy***

...moreover

**Question.** Can string stability be achieved with any other **linear controller**?

**Answer.** No, unless. . .

The objective of this lecture is to analyze the string stability of vehicle convoys for any linear controller based on **constant spacing policy**

Recall that string stability can be achieved with other spacing policy like, e.g., **constant time gap**



# Spacing error propagation in constant spacing policies

The results presented next are based on the following assumptions

## Assumptions

- 1 Identical vehicles modeled by  $G(s)$
- 2  $G(s)$  is linear, strictly proper, single-input-single-output and with two integrators
- 3 Identical control laws
- 4 Constant spacing policy

Hereafter,  $G(s)$  can be assumed as follows

$$G(s) = \frac{X(s)}{U(s)} = \frac{1}{ms^2(1 + s\tau)},$$

with  $X(s) = \mathcal{L}(x(t))$  and  $U(s) = \mathcal{L}(u(t))$

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## Predecessor following control scheme

Each vehicle longitudinal motion can be modeled as

$$X_i(s) = H(s) (U_i(s) + D_i(s)) + \frac{x_i(0)}{s}, \quad 1 \leq i \leq N,$$

where  $D_i(s)$  is an input disturbance and  $x_i(0) = -iL_{des}$ .

The *spacing error* is given by

$$\Delta_i(s) = X_{i-1}(s) - X_i(s) - \frac{L_{des}}{s}$$

Assume momentarily  $D_i(s) = 0$  and a local feedback control law based on the spacing error w.r.t. the predecessor. I.e.,  $U_i(s) = K(s)\Delta_i(s)$ .

Calculate  $\Delta_1(s)$  (drop the  $s$  argument)

$$\Delta_1 = X_0 - X_1 - \frac{L_{des}}{s}$$

# Predecessor following control scheme

*calculation of  $\Delta_1$  (cont.)*

$$\begin{aligned}\Delta_1 &= X_0 - HU_1 + \frac{L_{des}}{s} - \frac{L_{des}}{s} \\ &= X_0 - HK\Delta_1 \\ \Rightarrow \Delta_1 &= \underbrace{\frac{1}{1+HK}}_S X_0\end{aligned}$$

Calculate  $\Delta_i$

$$\begin{aligned}\Delta_i &= X_{i-1} - X_i - \frac{L_{des}}{s} \\ &= HK\Delta_{i-1} - (i-1)\frac{L_{des}}{s} - HK\Delta_i + i\frac{L_{des}}{s} - \frac{L_{des}}{s} \\ &= HK\Delta_{i-1} - HK\Delta_i \\ \Rightarrow \Delta_i &= \underbrace{\frac{HK}{1+HK}}_T \Delta_{i-1}, \quad i = 1, \dots, N\end{aligned}$$

# Predecessor following control scheme

## Remarks

- ❶ The transfer function from  $X_0$  to  $\Delta_1$  is the *sensitivity function*
- ❷ The transfer function from  $\Delta_{i-1}$  to  $\Delta_i$  is the *complementary sensitivity function*
- ❸ We would like to have
  - ❶  $|S(j\omega)|$  small for all frequencies (limiting the first spacing error)
  - ❷  $|T(j\omega)|$  small for all frequencies (limiting the error propagation)
- ❹ Classical trade-off between sensitivity and complementary sensitivity functions
- ❺ Given a  $K(s)$  stabilizing the closed-loop system,  $H(s)K(s)$  has two poles in the origin.

Hence,  $T(0) = 1$  and  $\|T\|_\infty \geq 1$

- ❻ Actually  $\|T\|_\infty > 1$  as shown next

# Predecessor following control scheme

## Theorem (analogous to Bode's integral formula)

Assume that the loop transfer function  $H(s)K(s)$  of a feedback system goes to zero faster than  $1/s$  as  $s \rightarrow \infty$  and let  $T(s)$  be the complementary sensitivity function. The complementary sensitivity function satisfies the following integral

$$\int_0^\infty \frac{\ln |T(j\omega)|}{\omega^2} d\omega = \pi \sum \frac{1}{z_i},$$

where  $z_i$  are right half-plane zeros.

Observe that,

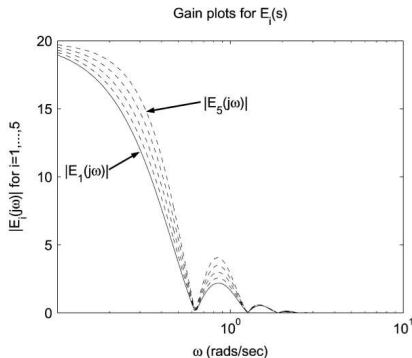
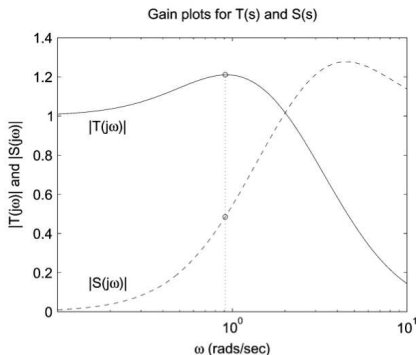
- ❶ Since  $H(s)$  is strictly proper,  $|T(j\omega)| \rightarrow 0$  as  $\omega \rightarrow \infty$ .  
Hence,  $\ln |T(j\omega)| < 0$  at high frequencies.
- ❷ The theorem implies that  $\ln |T(j\omega)| > 0$  for some frequency.  
Hence,  $|T(j\omega)| > 1$

## Example

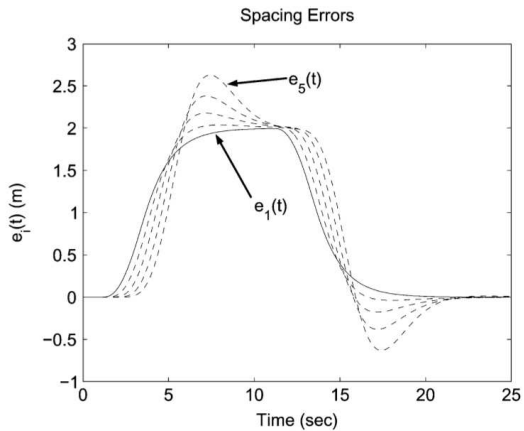
Consider the following vehicle model and controller

$$H(s) = \frac{1}{s^2(1 + 0.1s)}, \quad K(s) = \frac{1 + 2s}{1 + 0.05s}$$

Assume the lead vehicle accelerates from rest to 20 m/s over 12 s using the control input  $U_0(s) = \frac{1}{s^2} (e^{-s} - e^{-3s} - e^{-11s} + e^{-13s})$ , corresponding to a trapezoidal input



## Example (Cont.)





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# Predecessor and leader following control scheme

Modify the local control strategy in order to add info from the leader

$$U_i(s) = K_P(s)\Delta_i(s) + K_l(s) \left( X_0(s) - X_i(s) - i \frac{L_{des}}{s} \right)$$

Calculate  $\Delta_1$

$$\begin{aligned} \Delta_1 &= X_0 - X_1 - \frac{L_{des}}{s} \\ &= HU_1 - \frac{L_{des}}{s} + \frac{L_{des}}{s} - \frac{L_{des}}{s} \\ &= X_0 - HK_P\Delta_1 - HK_l \underbrace{\left( X_0 - X_1 - \frac{L_{des}}{s} \right)}_{\Delta_1} \\ \Rightarrow \Delta_1 &= \underbrace{\frac{1}{1 + H(K_P + K_l)}}_{S_{lp}(s)} X_0 \end{aligned}$$

# Predecessor and leader following control scheme

Calculate  $\Delta_i$

$$\begin{aligned}\Delta_i &= X_{i-1} - X_i - \frac{L_{des}}{s} \\&= HU_{i-1} - (i-1)\frac{L_{des}}{s} - HU_i + i\frac{L_{des}}{s} - \frac{L_{des}}{s} \\&= HK_P\Delta_{i-1} + HK_l \left[ X_0 - X_{i-1} - (i-1)\frac{L_{des}}{s} \right] \\&\quad - HK_P\Delta_i - HK_l \left[ X_0 - X_i - i\frac{L_{des}}{s} \right] \\&= HK_P\Delta_{i-1} - HK_P\Delta_i - HK_l \underbrace{\left[ X_{i-1} + (i-1)\frac{L_{des}}{s} - X_i - i\frac{L_{des}}{s} \right]}_{\Delta_i} \\&\Rightarrow \Delta_i = \underbrace{\frac{HK_P}{1 + H(K_P + K_l)}}_{T_{lp}(s)} \Delta_{i-1}\end{aligned}$$

## Predecessor and leader following control scheme

Clearly,  $K_P$  and  $K_I$  can now be easily designed in order to guarantee  $\|T_{lp}\|_\infty < 1$ .

For example, if we chose  $K_P(s) = K_I(s)$  then  $T_{lp}(0) = 0.5$ .

Consider the case of the previous example, if we chose  $K_P(s) = K_I(s) = 1/2K(s)$  then  $T_{lp}(s) = 1/2T(s)$ . In the predecessor following scheme,  $\|T\|_\infty = 1.21$  while  $\|T_{pl}\|_\infty = 0.605$ .

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# Disturbance propagation

Recall we have modeled each vehicle as

$$X_i(s) = H(s) (U_i(s) + D_i(s)) + \frac{x_i(0)}{s}, \quad 1 \leq i \leq N,$$

Now we analyze the effects of the disturbance  $D_i$  on the stability of the string

Calculate  $\Delta_1$  and  $\Delta_i$

$$\begin{aligned} \Delta_1 &= X_0 - X_1 - \frac{L_{des}}{s} \\ &= X_0 - HD_1 - HU_1 + \frac{L_{des}}{s} - \frac{L_{des}}{s} \\ &\Rightarrow \underline{\Delta_1 = X_0 - HD_1 - HU_1} \end{aligned}$$

$$\begin{aligned} \Delta_i &= X_{i-1} - X_i - \frac{L_{des}}{s} \\ &= HD_{i-1} + HU_{i-1} - HD_i - HU_i - (i-1)\frac{L_{des}}{s} + i\frac{L_{des}}{s} - \frac{L_{des}}{s} \\ &\Rightarrow \underline{\Delta_i = HD_{i-1} - HD_i + HU_{i-1} - HU_i} \end{aligned}$$

# Disturbance propagation

Introduce the vectors

$$\bar{\Delta} = [\Delta_1, \dots, \Delta_N], \quad \bar{D} = [D_1, \dots, D_N], \quad \bar{U} = [U_1, \dots, U_N]$$

The spacing error dynamics can be compactly rewritten as

$$\bar{\Delta} = P_{11} \begin{bmatrix} X_0 \\ \bar{D} \end{bmatrix} + P_{12} \bar{U},$$

with

$$P_{11} = \begin{bmatrix} 1 & -H & & & \\ 0 & H & -H & & \\ \vdots & & \ddots & \ddots & \\ 0 & & & H & -H \end{bmatrix}, \quad P_{12} = \begin{bmatrix} -H & & & & \\ H & -H & & & \\ & & \ddots & \ddots & \\ & & & H & -H \end{bmatrix}$$

# Disturbance propagation

The control law  $U_i(s) = K_P(s)\Delta_i(s) + K_l(s) \left( X_0(s) - X_i(s) - i\frac{L_{des}}{s} \right)$  can be rewritten as

$$\begin{aligned} U_i(s) &= K_P(s)\Delta_i(s) + K_l(s) \left( X_0(s) - X_1(s) - \frac{L_{des}}{s} \right. \\ &\quad \left. + X_1(s) - X_2(s) - i\frac{L_{des}}{s} \dots + X_{i-1}(s) - X_i(s) - i\frac{L_{des}}{s} \right) \\ &= K_P(s)\Delta_i(s) + K_l(s) \sum_{k=1}^i \Delta_k \end{aligned}$$

Hence,  $\bar{U} = \bar{K}\bar{\Delta}$  with

$$\bar{K} = \begin{bmatrix} K_l + K_P & & & \\ K_l & K_l + K_P & & \\ \vdots & \ddots & \ddots & \\ K_l & \dots & K_l & K_l + K_P \end{bmatrix}$$



# Disturbance propagation

By substituting  $\bar{U}$  in  $\bar{\Delta} = P_{11} \begin{bmatrix} X_0 \\ \bar{D} \end{bmatrix} + P_{12}\bar{U}$ , write the error vector as

$$\bar{\Delta} = \begin{bmatrix} 1 \\ T_{lp} \\ \vdots \\ T_{lp}^{N-1} \end{bmatrix} S_{lp} X_0 - S_{lp} H + \begin{bmatrix} 1 & & & \\ & T_{lp} - 1 & 1 & \\ & \vdots & \ddots & \ddots \\ (T_{lp} - 1) T_{lp}^{N-2} & \dots & T_{lp} - 1 & 1 \end{bmatrix} \bar{D}$$

Contribution from the first term falls in the analysis done for the *leader and predecessor following scheme*. Focus on the second term

# Disturbance propagation

Consider the matrix

$$\begin{bmatrix} 1 & & & & \\ T_{lp} - 1 & 1 & & & \\ \vdots & \ddots & \ddots & & \\ (T_{lp} - 1) T_{lp}^{N-2} & \dots & T_{lp} - 1 & 1 & \end{bmatrix}$$

Recall that  $T_{lp} = \frac{H K_P}{1 + H (K_P + K_l)}$

Distinguish the following two cases

- ❶  $K_P = 0$ . I.e., only the information from the leader is used. In this case  $T_{lp} = 0$ . There is not disturbance propagation and  $D_i$  affects  $\Delta_{i+1}$  through  $S_{lp}H$
- ❷  $K_l = 0$ . I.e., only the information from the preceding vehicle is used. We already know that  $|T_{lp}(j\omega)| > 1$  for some  $\omega$ . Moreover, disturbances propagate and  $D_i$  affects  $\Delta_{i+1}$  through  $(T_{lp} - 1) S_{lp}H$ . On the other hand  $|(T_{lp} - 1)| \ll 0$  at low frequencies and the disturbances are attenuated.

# Disturbance propagation

## In conclusion

- 1 With leader information only there is not disturbance propagation.
- 2 Information from the predecessor reduces the effect of the disturbance on the spacing error.

More rigorously,

## Theorem

Assume  $H$  has two poles in the origin and the closed loop is stable. Be  $\bar{G}_{d\delta}(s)$  the transfer matrix from the disturbance to the spacing errors.

- 1 If  $\|T_{lp}\|_{\infty} > 1$ , then given any  $M > 0$ , there exists a  $N$  (platoon length) such that  $\|\bar{G}_{d\delta}\|_{\infty} > M$
- 2 If  $\|T_{lp}\|_{\infty} < 1$ , then there exists a  $M > 0$ , such that  $\|\bar{G}_{d\delta}\|_{\infty} \leq M, \forall N$

# Disturbance propagation

## Remarks on the Theorem

- 1 If  $\|T_{lp}\|_{\infty} > 1$  (e.g., in a predecessor following scheme) the gain from the disturbance to the spacing errors grows with the platoon length
- 2 If  $\|T_{lp}\|_{\infty} < 1$  (e.g., in a predecessor and leader following scheme) the gain from the disturbance to the spacing errors is bounded as the platoon length increases

The predecessor and leader following scheme is scalable. Nevertheless *communication is needed*.

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# String stability in heterogenous platoons

The results presented so far are based on the following assumptions

## Assumptions

- ❶ **Identical vehicles modeled by  $G(s)$**
- ❷  $G(s)$  is linear, strictly proper, single-input-single-output and with two integrators
- ❸ **Identical control laws**
- ❹ Constant spacing policy

We now remove Assumptions 1 and 3

For simplicity, we assume the platoon is made of three types of vehicles, with *slow*, *medium* and *fast* dynamics

**Objective:** analyze the string stability of the predecessor and the predecessor-leader following schemes

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## Heterogenous platoons. Predecessor following scheme

Calculate  $\Delta_1$  and  $\Delta_i$

$$\begin{aligned}\Delta_1 &= X_0 - X_1 - \frac{L_{des}}{s} \\ &= X_0 - H_1 U_1 + \frac{L_{des}}{s} - \frac{L_{des}}{s} = X_0 - H_1 K_1 \Delta_1 \\ \Rightarrow \Delta_1 &= \underbrace{\frac{1}{1 + H_1 K_1}}_{S_1} X_0\end{aligned}$$

$$\begin{aligned}\Delta_i &= X_{i-1} - X_i - \frac{L_{des}}{s} \\ &= H_{i-1} K_{i-1} \Delta_{i-1} - H_i K_i \Delta_i - (i-1) \frac{L_{des}}{s} + i \frac{L_{des}}{s} - \frac{L_{des}}{s} \\ \Rightarrow \Delta_i &= \underbrace{\frac{H_{i-1} K_{i-1}}{1 + H_i K_i}}_{\tilde{T}_{i-1}} \Delta_{i-1}, \quad i = 1, \dots, N\end{aligned}$$



# Heterogenous platoons. Predecessor following scheme

From

$$\Delta_1 = \underbrace{\frac{1}{1 + H_1 K_1}}_{S_1} X_0, \quad \Delta_i = \underbrace{\frac{H_{i-1} K_{i-1}}{1 + H_i K_i}}_{\tilde{T}_{i-1}} \Delta_{i-1}, \quad i = 1, \dots, N,$$

rewrite the error dynamics as

$$\begin{aligned} \Delta_i &= \frac{H_{i-1} K_{i-1}}{1 + H_i K_i} \Delta_{i-1} \\ &= \frac{H_{i-1} K_{i-1}}{1 + H_i K_i} \cdot \frac{H_{i-2} K_{i-2}}{1 + H_{i-1} K_{i-1}} \Delta_{i-2} \\ &= \frac{H_{i-1} K_{i-1}}{1 + H_i K_i} \cdot \frac{H_{i-2} K_{i-2}}{1 + H_{i-1} K_{i-1}} \cdot \dots \cdot \frac{1}{1 + H_1 K_1} \\ &= \frac{1}{1 + H_i K_i} \cdot \frac{H_{i-1} K_{i-1}}{1 + H_{i-1} K_{i-1}} \cdot \dots \cdot \frac{H_1 K_1}{1 + H_1 K_1} X_0 \\ &= S_i \left( \prod_{k=1}^{i-1} T_k \right) X_0 \end{aligned}$$

# Heterogenous platoons. Predecessor following scheme

In

$$\Delta_i = S_i \left( \prod_{k=1}^{i-1} T_k \right) X_0,$$

$S_i$  and  $T_i$  are the sensitivity and complementary sensitivity functions of the  $i$ -th vehicle.

Recall that  $\|T_i\|_\infty > 1$ . Hence  $\left\| \prod_{k=1}^{i-1} T_k \right\|_\infty$  is unbounded as  $N$  grows

More in details, from

$$\Delta_i = \frac{H_{i-1}K_{i-1}}{1 + H_iK_i} \Delta_{i-1},$$

it follows that

- 1 if the preceding vehicle has faster dynamics  $\|\tilde{T}_i\|_\infty > 1$  and the error will grow
- 2 if the preceding vehicle has slower dynamics  $\|\tilde{T}_i\|_\infty < 1$  and the error will shrink

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# Heterogenous platoons. Predecessor-leader following

Consider the control law

$$U_i(s) = K_{P,i}(s)\Delta_i(s) + K_{l,i}(s) \left( X_0(s) - X_i(s) - i \frac{L_{des}}{s} \right),$$

where  $K_{P,i} = p_i K_i$  and  $K_{l,i} = l_i K_i$ ,  $K_i$  is the low level controller of the  $i$ -th vehicle,  $p_i$ ,  $l_i$  are positive scalar smaller than 1 and such that  $p_i + l_i = 1$

Calculate  $\Delta_1$  and  $\Delta_i$

$$\Delta_1 = \frac{1}{\underbrace{1 + H_1 K_1}_{S_1}} X_0$$

$$\Delta_i = \hat{T}_{i-1} \Delta_{i-1} + A_{i-1} \sum_{k=1}^{i-1} E_k, \quad i = 2, \dots, N$$

## Heterogenous platoons. Predecessor-leader following

$\hat{T}_i$  and  $A_i$  are defined as

$$\begin{aligned}\hat{T}_i &= \frac{H_i K_{P,i}}{1 + H_{i+1} (K_{P,i+1} + K_{l,i+1})} \\ A_i &= \frac{H_i K_{l,i} - H_{i+1} K_{l,i+1}}{1 + H_{i+1} (K_{P,i+1} + K_{l,i+1})}\end{aligned}$$

We have seen that heterogenous string stable (unstable) platoons do not attenuate (amplify) the spacing errors uniformly (i.e.,  $\|\delta_i\|_\infty < \|\delta_{i-1}\|_\infty$ )

Hence, we have to define a stable string of heterogenous vehicles

### Definition

A heterogenous vehicle string is string stable if the propagating errors stay uniformly bounded for all string lengths and vehicle type orderings

# Heterogenous platoons. Predecessor-leader following

Define the transfer functions

$$\begin{aligned}T_{p,x} &= \frac{H_x K_{P,x}}{1 + H_x (K_{P,x} + K_{l,x})}, \\T_{l,x} &= \frac{H_x K_{l,x}}{1 + H_x (K_{P,x} + K_{l,x})}, \\x &\in \{s, m, f\}\end{aligned}$$

Rewrite the error dynamics as

$$\begin{aligned}\Delta_i &= F_i X_0 \\F_1 &= 1 - T_{p,x_1} - T_{l,x_1}, \\F_i &= \prod_{j=1}^{i-1} T_{p,x_j} - \prod_{j=1}^i T_{p,x_j} \\&\quad + (1 - T_{p,x_i}) \left( \left( \sum_{k=1}^{i-2} \left( \prod_{j=k+1}^{i-1} T_{p,x_j} \right) T_{l,x_k} \right) + T_{l,x_{i-1}} \right) - T_{l,x_i}\end{aligned}$$

# Heterogenous platoons. Predecessor-leader following

## Remark

Clearly, if there exists a finite  $M > 0$  such that  $\|F_i\|_\infty \leq M \forall i$  and vehicle type orderings, then  $\|\Delta_i\|_\infty$  stays uniformly bounded for all  $i$  and vehicle type orderings. I.e., the heterogenous platoon is string stable according to the definition

The following result holds

## Theorem

Assume  $H(s)$  has two poles in the origin and the closed-loop is stable. There exists a finite  $M > 0$  such that  $\|F_i\|_\infty \leq M \forall i$  and vehicle type orderings *if and only if*  $\|T_{p,x}\|_\infty < 1$  for all vehicle types.

# Heterogenous platoons. Predecessor-leader following

## Guidelines for controller design

- 1 Design the local controller for each vehicle such that  $\|T_{p,x}\|_\infty < 1$
- 2 In order to bound the errors in such a way controllers do not have to be redesigned when vehicle ordering and/or platoon length change a conservative design is needed
- 3 In case of homogenous platoon, the largest error is the first  $\Delta_1 = S_1 X_0$ . The largest possible  $\Delta_1$  is when the first follower is a *slow* vehicle. In this case  $|S_1|$  is the largest at low frequencies.
- 4 The local controllers can then be designed in order to achieve  $\|e_i\|_\infty \leq \|e_1\|_\infty, \forall i$